

# PID Example: DC Motor Speed Control

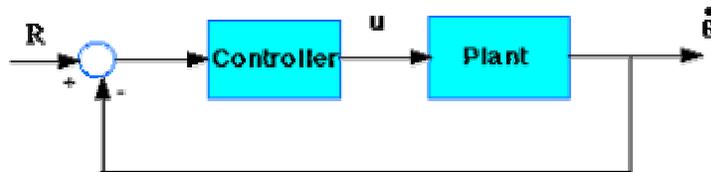
From the main problem, the dynamic equations and the open-loop transfer function of the DC Motor are:

$$s(Js + b)\Theta(s) = KI(s)$$

$$(Ls + R)I(s) = V - Ks\Theta(s)$$

$$\frac{\Theta}{V} = \frac{K}{(Js + b)(Ls + R) + K^2}$$

and the system schematic looks like:



With a 1 rad/sec step input, the design criteria are:

- Settling time less than 2 seconds
- Overshoot less than 5%
- Steady-state error less than 1%

Now let's design a PID controller and add it into the system. First create a new m-file and type in the following commands

```
J=0.01;
b=0.1;
K=0.01;
R=1;
L=0.5;
num=K;
den=[(J*L) ((J*R)+(L*b)) ((b*R)+K^2)];
```

Recall that the transfer function for a PID controller is:

$$K_p + \frac{K_i}{s} + K_d s = \frac{K_D s^2 + K_P s + K_I}{s}$$

# Proportional control

Let's first try using a proportional controller with a gain of 100. Add the following code to the end of your m-file:

```
Kp=100;  
numa=Kp*num;  
dena=den;
```

To determine the closed-loop transfer function, we use the `cloop` command. Add the following line to your m-file:

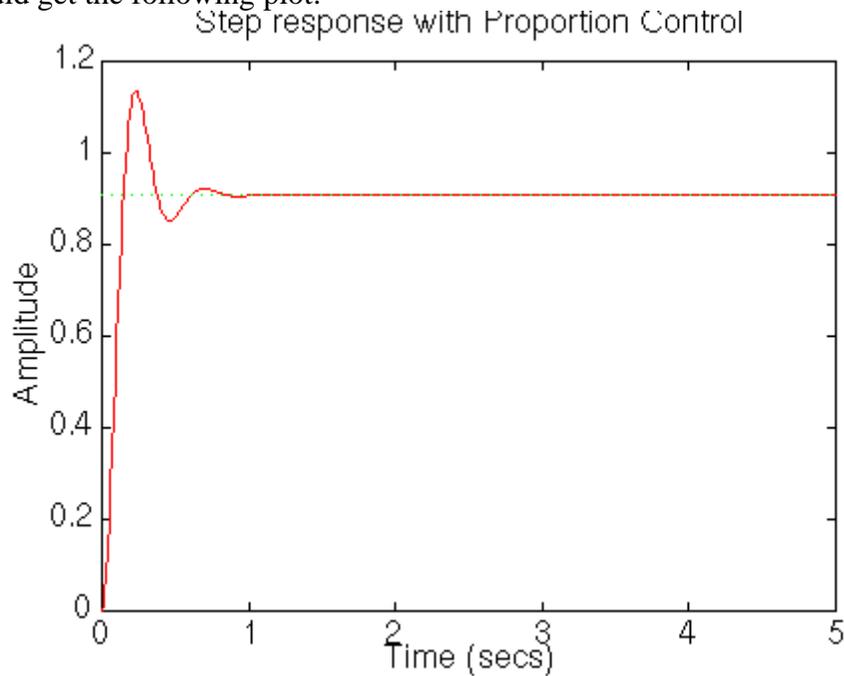
```
[numac,denac]=cloop(numa,dena);
```

Note that `numac` and `denac` are the numerator and the denominator of the overall closed-loop transfer function.

Now let's see how the step response looks, add the following to the end of your m-file, and run it in the command window:

```
t=0:0.01:5;  
step(numac,denac,t)  
title('Step response with Proportion Control')
```

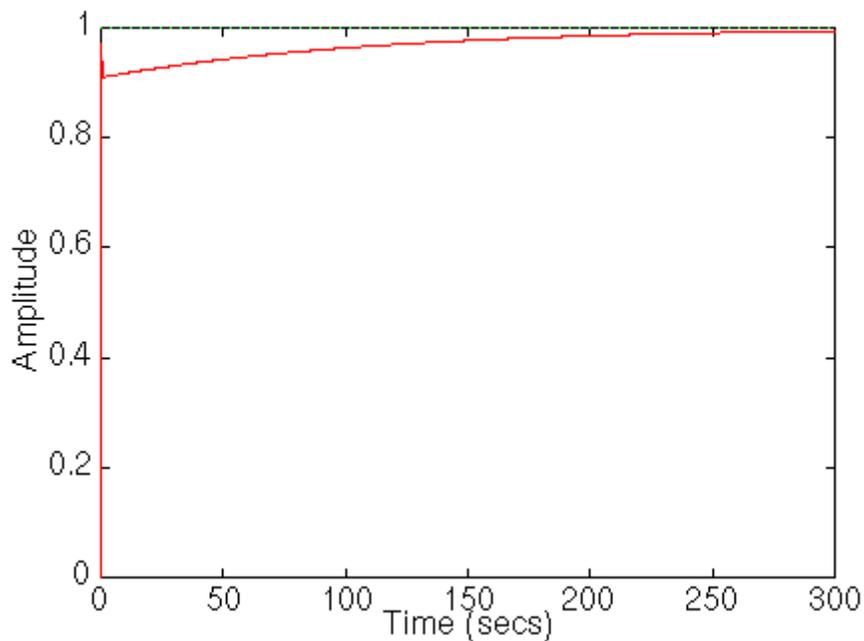
You should get the following plot:



## PID control

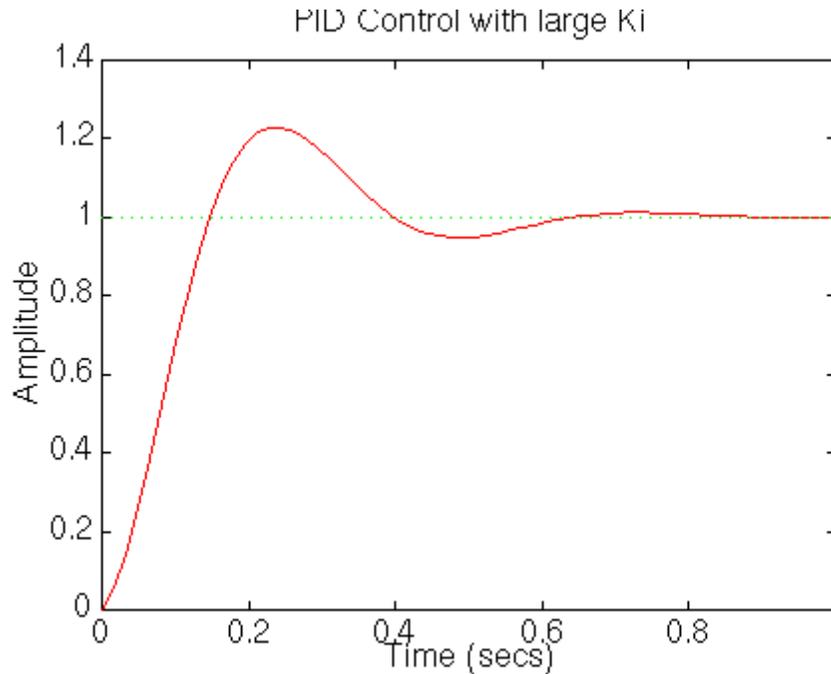
From the plot above we see that both the steady-state error and the overshoot are too large. Let's try a PID controller with small  $K_i$  and  $K_d$ . Change your m-file so it looks like the following. Running this new m-file gives you the following plot.

```
J=0.01;  
b=0.1;  
K=0.01;  
R=1;  
L=0.5;  
num=K;  
den=[(J*L) ((J*R)+(L*b)) ((b*R)+K^2)];  
  
Kp=100;  
Ki=1;  
Kd=1;  
numc=[Kd, Kp, Ki];  
denc=[1 0];  
numa=conv(num,numc);  
dena=conv(den,denc);  
[numac,denac]=cloop(numa,dena);  
step(numac,denac)  
title('PID Control with small Ki and Kd')
```

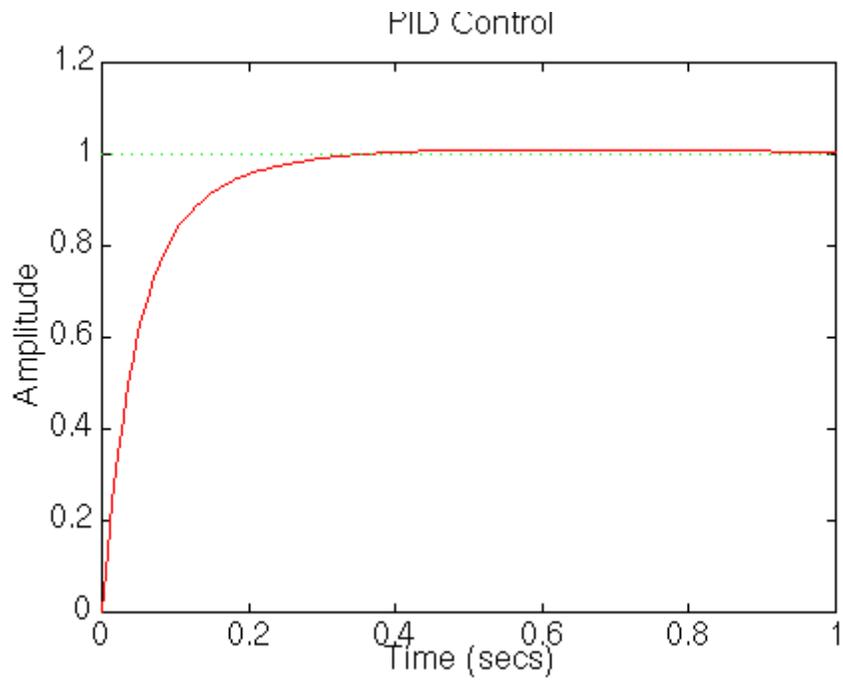


## Tuning the gains

Now the settling time is too long. Let's increase  $K_i$  to reduce the settling time. Go back to your m-file and change  $K_i$  to 200. Rerun the file and you should get the plot like this:



Now we see that the response is much faster than before, but the large  $K_i$  has worsened the transient response (big overshoot). Let's increase  $K_d$  to reduce the overshoot. Go back to the m-file and change  $K_d$  to 10. Rerun it and you should get this plot:



So now we know that if we use a PID controller with

$$\begin{aligned} K_p &= 100, \\ K_i &= 200, \\ K_d &= 10, \end{aligned}$$

all of our design requirements will be satisfied.